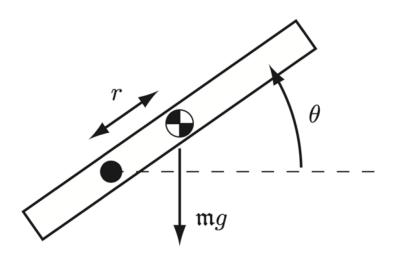
Where we are:

- Chap 2 Configuration Space
- Chap 3 Rigid-Body Motions
- Chap 4 Forward Kinematics
- Chap 5 Velocity Kinematics and Statics
- Chap 6 Inverse Kinematics
- Chap 8 Dynamics of Open Chains
- Chap 9 Trajectory Generation
- Chap 11 Robot Control
 - 11.1 Control System Overview
 - 11.2 Error Dynamics
 - 11.3 Motion Control with Velocity Inputs
 - 11.4 Motion Control with Torque or Force Inputs

Chap 13 Wheeled Mobile Robots

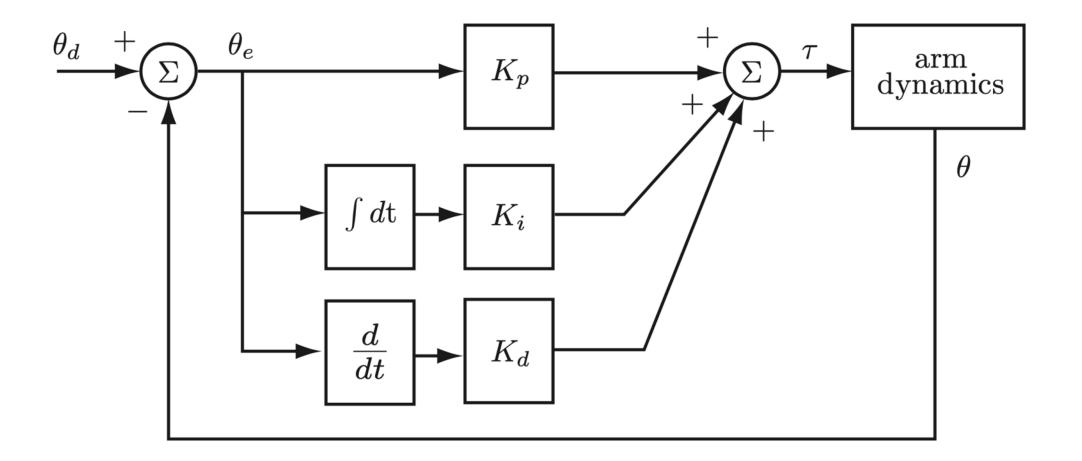


$$\tau = M\ddot{\theta} + \mathfrak{m}gr\cos\theta + b\dot{\theta}$$

$$\tau = M\ddot{\theta} + h(\theta, \dot{\theta})$$

Proportional-Integral-Derivative (PID) control

$$\tau = K_p \theta_e + K_i \int \theta_e(\mathbf{t}) d\mathbf{t} + K_d \dot{\theta}_e$$



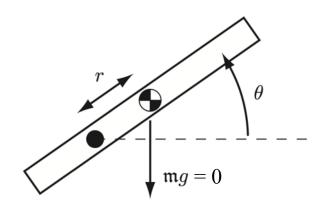
Setpoint PD control, g = 0

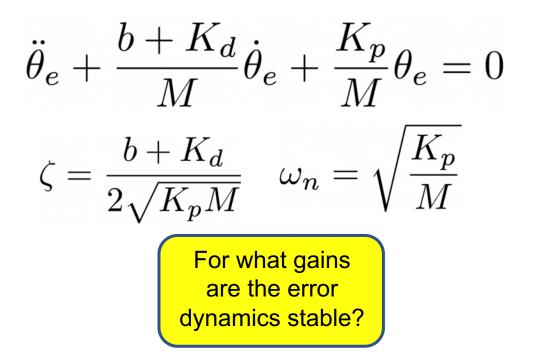
$$\tau = K_p \theta_e + K_i \int \theta_e(t) dt + K_d \dot{\theta}_e$$

$$\tau = M \ddot{\theta} + \mathfrak{m} gr \cos \theta + b \dot{\theta}$$

$$\dot{\theta}_d = \ddot{\theta}_d = 0$$

$$M\ddot{\theta} + b\dot{\theta} = K_p\theta_e + K_d\dot{\theta}_e$$





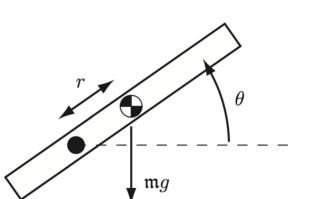
Setpoint PD control, $g \neq 0$

$$\tau = K_p \theta_e + K_i \int \theta_e(t) dt + K_d \dot{\theta}_e$$

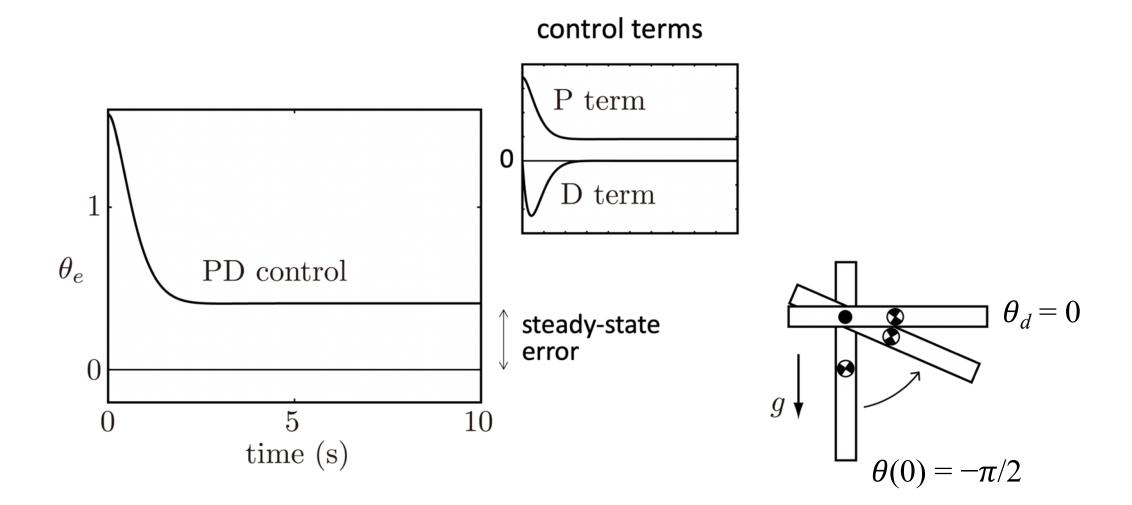
$$\tau = M \ddot{\theta} + \mathfrak{m} gr \cos \theta + b \dot{\theta}$$

$$\dot{\theta}_d = \ddot{\theta}_d = 0$$

$$M \ddot{\theta}_e + (b + K_d) \dot{\theta}_e + K_p \theta_e = \mathfrak{m} gr \cos \theta$$



Nonhomogeneous. What is the steadystate error?



Setpoint PID control, $g \neq 0$

$$\tau = K_p \theta_e + K_i \int \theta_e(t) dt + K_d \dot{\theta}_e$$

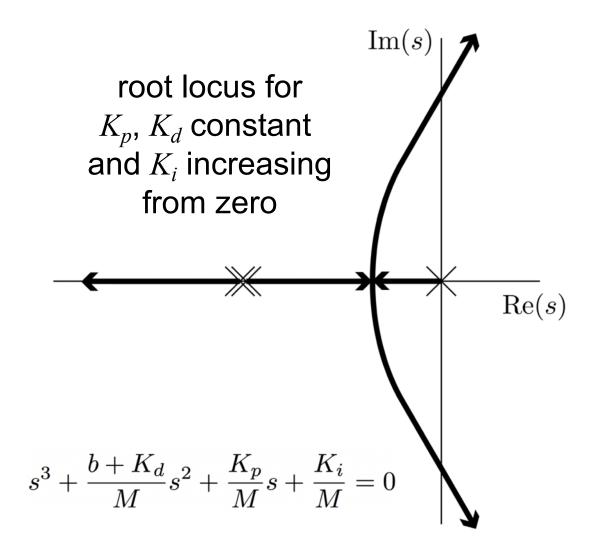
$$\tau = M \ddot{\theta} + \mathfrak{m} gr \cos \theta + b \dot{\theta}$$

$$\dot{\theta}_d = \ddot{\theta}_d = 0$$

$$\tau_{\text{dist}}$$

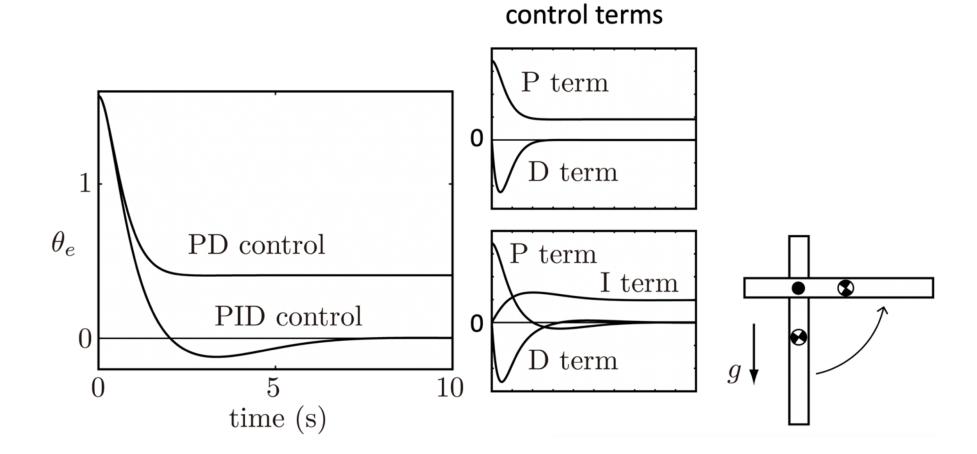
$$M \ddot{\theta} + (b + K_i) \dot{\theta} + K_i \theta + K_i \int \theta_i$$

$$M\theta_e + (b + K_d)\theta_e + K_p\theta_e + K_i \int \theta_e(t)dt = \tau_{\text{dist}}$$
$$M\theta_e^{(3)} + (b + K_d)\ddot{\theta}_e + K_p\dot{\theta}_e + K_i\theta_e = \mathbf{0}$$
$$s^3 + \frac{b + K_d}{M}s^2 + \frac{K_p}{M}s + \frac{K_i}{M} = 0$$



$$\begin{array}{rcl}
K_d &> & -b \\
K_p &> & 0 \\
\hline
(b+K_d)K_p \\
M &> & K_i &> & 0
\end{array}$$

 K_i improves steady-state response but can worsen the transient response.

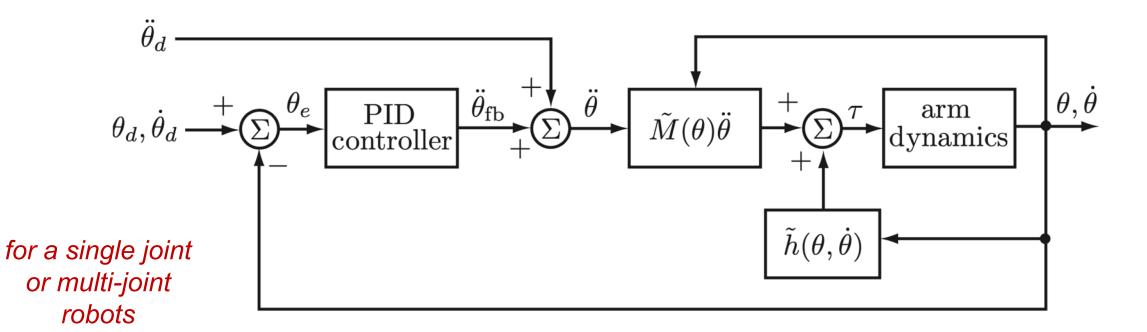


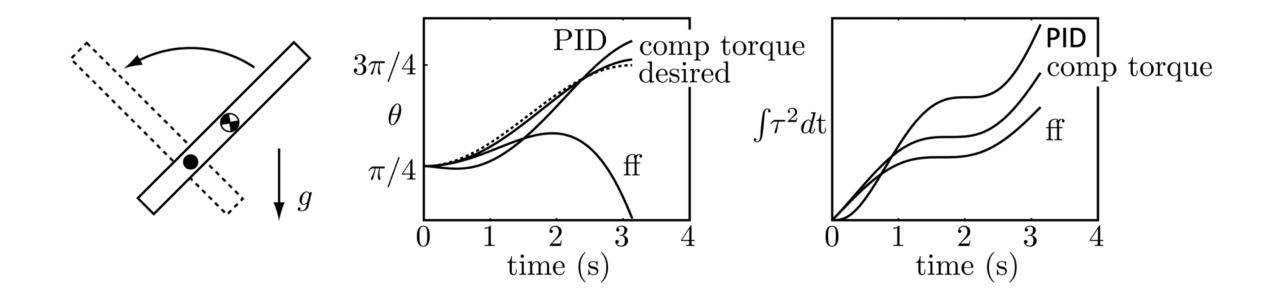
What about tracking general trajectories, not just setpoint control?

$$\tau = M \left(\ddot{\ddot{\theta}}_d + K_p \theta_e + K_i \int \theta_e dt + K_d \dot{\theta}_e \right) + h(\theta, \dot{\theta})$$
$$\ddot{\theta}_e = \ddot{\theta}_d - \ddot{\theta}$$
$$\ddot{\theta}_e = -K_d \dot{\theta}_e - K_p \theta_e - K_i \int \theta_e dt$$
$$\theta_e^{(3)} + K_d \ddot{\theta}_e + K_p \dot{\theta}_e + K_i \theta_e = 0$$

Computed torque control (feedback linearization)

$$\tau = \tilde{M}(\theta) \left(\ddot{\theta}_d + K_p \theta_e + K_i \int \theta_e(\mathbf{t}) d\mathbf{t} + K_d \dot{\theta}_e \right) + \tilde{h}(\theta, \dot{\theta})$$





Task-space computed torque control

 $\mathcal{F}_b = \Lambda(\theta)\mathcal{V}_b + \eta(\theta,\mathcal{V}_b)$ dynamic model: $\{\tilde{\Lambda}, \tilde{\eta}\}$ $\tau = \tilde{M}(\theta) \left(\ddot{\theta}_d + K_p \theta_e + K_i \int \theta_e dt + K_d \dot{\theta}_e \right) + \tilde{h}(\theta, \dot{\theta})$ $\mathcal{F}_{b} = \tilde{\Lambda}(\theta) \left(\dot{\mathcal{V}}_{d} + K_{p}X_{e} + K_{i} \int X_{e}dt + K_{d}\mathcal{V}_{e} \right) + \tilde{\eta}(\theta, \mathcal{V}_{b})$ $[X_e] = \log(X^{-1}X_d)$ $\mathcal{V}_e = [\operatorname{Ad}_{X^{-1}X_{\mathsf{J}}}]\mathcal{V}_d - \mathcal{V}_{\mathsf{h}}$ $au = J_b^{\mathrm{T}}(\theta)\mathcal{F}_b$

What if your dynamic model is poor?

The characteristic equation of the error dynamics are

$$s^5 + 2s^4 + s^3 + 2s^2 + 4s + 2 = 0$$

Write the error dynamics in the form $\dot{x} = Ax$. Determine if the system is stable. (Use any software you want.)