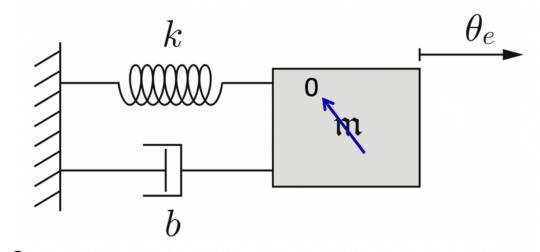
#### Where we are:

Chap 2	Configuration Space
Chap 3	Rigid-Body Motions
Chap 4	Forward Kinematics
Chap 5	Velocity Kinematics and Statics
Chap 6	Inverse Kinematics
Chap 8	Dynamics of Open Chains
Chap 9	Trajectory Generation
Chap 11	Robot Control
•	11.1 Control System Overview
	11.2 Error Dynamics
Chap 13	Wheeled Mobile Robots

#### First-order error dynamics



$$\dot{\theta}_e + b\dot{\theta}_e + k\theta_e = \mathbf{0}$$
 
$$\dot{\theta}_e(t) + \frac{k}{b}\theta_e(t) = 0$$

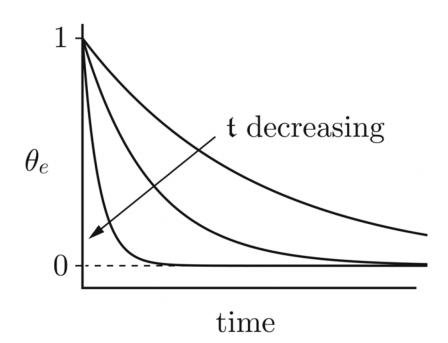
#### standard first-order form

# time constant

$$\mathfrak{t} = b/k$$

$$\dot{\theta}_e(t) + \frac{1}{t}\theta_e(t) = 0$$

#### **First-order error dynamics**



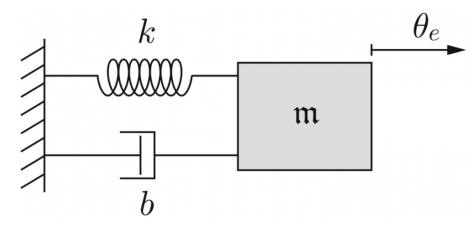
$$\theta_e(t) = e^{-t/\mathfrak{t}}\theta_e(0)$$

$$\theta_e(0) = 1$$

$$\frac{\theta_e(t)}{\theta_e(0)} = 0.02 = e^{-t/\mathfrak{t}}$$

$$\ln 0.02 = -t/\mathfrak{t} \quad \to \quad t = 3.91\mathfrak{t}$$

#### **Second-order error dynamics**



$$\ddot{\theta}_e(t) + \frac{b}{\mathfrak{m}}\dot{\theta}_e(t) + \frac{k}{\mathfrak{m}}\theta_e(t) = 0$$

#### natural frequency damping ratio

$$\omega_n = \sqrt{k/\mathfrak{m}}$$
  $\zeta = b/(2\sqrt{k\mathfrak{m}})$ 

$$\omega_n = \sqrt{k/\mathfrak{m}} \qquad \zeta = b/(2\sqrt{k\mathfrak{m}})$$
$$\ddot{\theta}_e(t) + 2\zeta\omega_n\dot{\theta}_e(t) + \omega_n^2\theta_e(t) = 0$$

standard second-order form

#### **Second-order error dynamics**

$$\ddot{\theta}_e(t) + 2\zeta\omega_n\dot{\theta}_e(t) + \omega_n^2\theta_e(t) = 0$$

$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

$$s_{1,2} = -\zeta\omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$

 $\zeta > 1$ : Overdamped

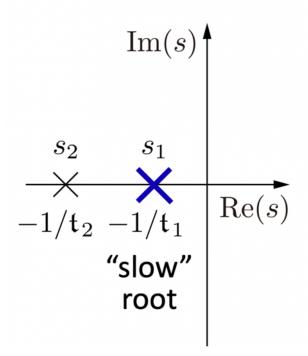
 $\zeta = 1$ : Critically damped

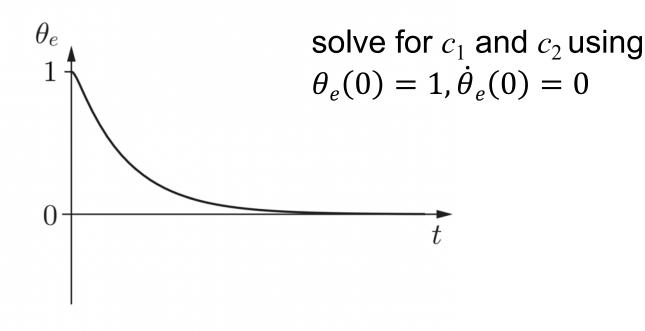
 $\zeta < 1$ : Underdamped

$$\zeta > 1$$
: Overdamped

$$\theta_e(t) = c_1 e^{s_1 t} + c_2 e^{s_2 t}$$

$$s_1 = -\zeta \omega_n + \omega_n \sqrt{\zeta^2 - 1}$$
$$s_2 = -\zeta \omega_n - \omega_n \sqrt{\zeta^2 - 1}$$

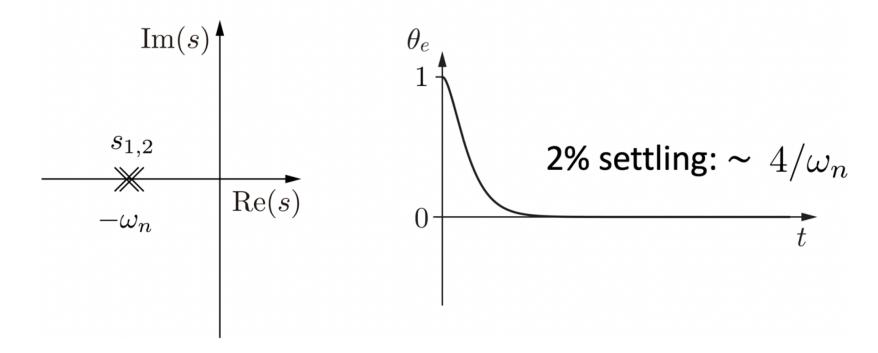




# $\zeta=1$ : Critically damped

$$\theta_e(t) = (c_1 + c_2 t)e^{-\omega_n t}$$

$$s_{1,2} = -\omega_n$$



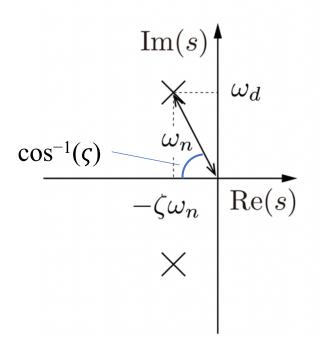
# $\zeta < 1$ : Underdamped

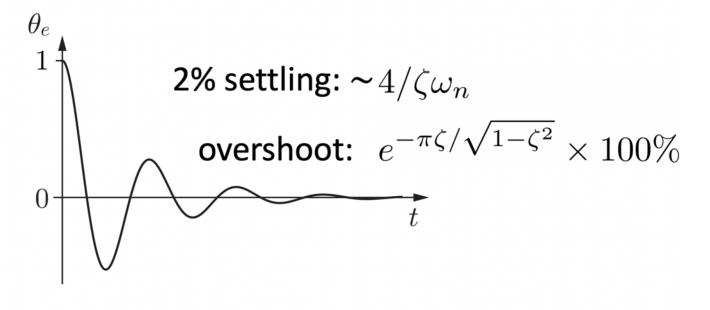
$$\theta_e(t) = (c_1 \cos \omega_d t + c_2 \sin \omega_d t) e^{-\zeta \omega_n t}$$

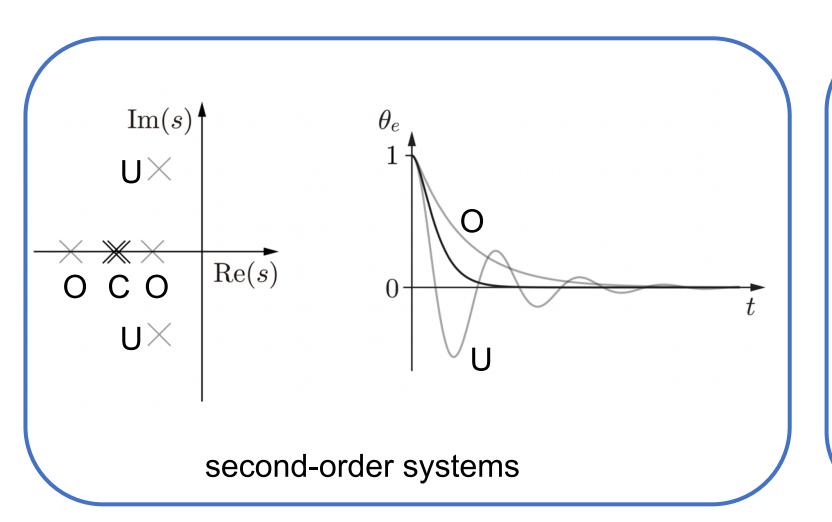
#### damped natural frequency

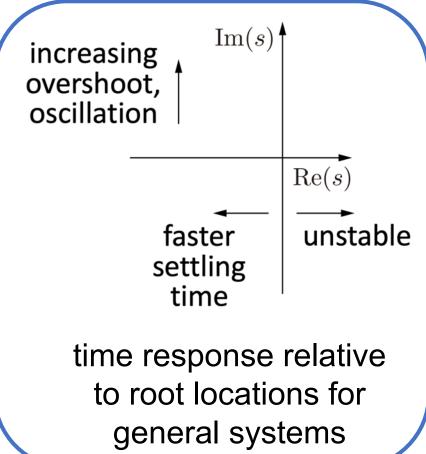
$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

$$s_{1,2} = -\zeta \omega_n \pm j\omega_d$$









$$\omega_n = \sqrt{k/\mathfrak{m}} \qquad \zeta = b/(2\sqrt{k\mathfrak{m}})$$

2% settling:  $\sim 4/\zeta \omega_n$ 

$$\ddot{\theta}_e(t) + 2\zeta\omega_n\dot{\theta}_e(t) + \omega_n^2\theta_e(t) = 0$$

overshoot:  $e^{-\pi\zeta/\sqrt{1-\zeta^2}} \times 100\%$ 

When controlling a robot joint, what do b, k, and m usually correspond to?

How do you change  $\mathfrak{m}$  to decrease settling time? k, b?

How do you change m to decrease overshoot? k, b?

Shown are one of the roots of five different second-order systems, A, B, C, D, and E. List them in the following orders:

- Natural frequency, highest to lowest.
- 2. Damped natural frequency, highest to lowest.
- 3. Damping ratio, highest to lowest.
- 4. Overshoot in unit step error response, highest to lowest.
- 5. Settling time, longest to shortest.

Which has the "best" transient response?

